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(54) FLOW CONTROL SYSTEM

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- (51) **Int. Cl.** *F17C 5/06* (2006.01) *F17C 7/00* (2006.01) *F17C 13/04* (2006.01)
- (52) U.S. Cl.

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See application file for complete search history.

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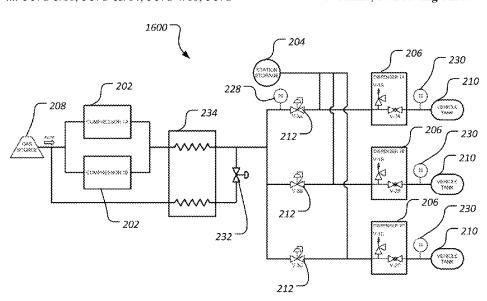
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(57) ABSTRACT

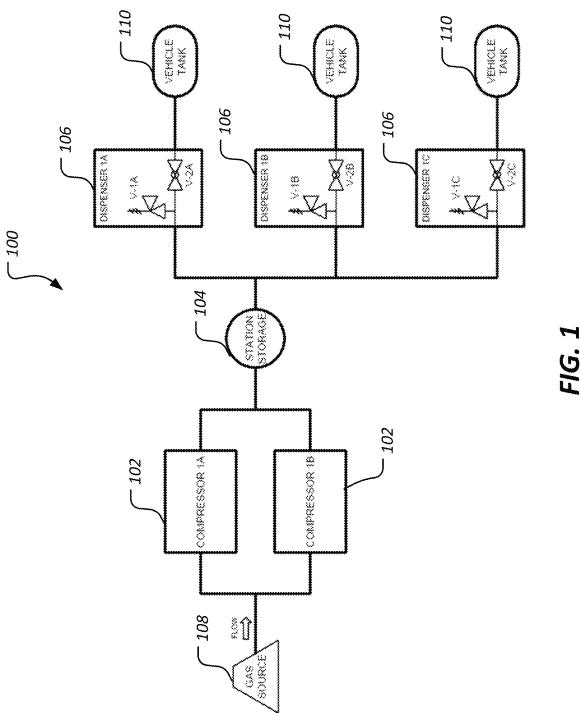
A compressed natural gas (CNG) refueling station system includes a compressor, a dispenser, and at least one of a valve and an orifice disposed in fluid communication between the compressor and the dispenser.

8 Claims, 16 Drawing Sheets



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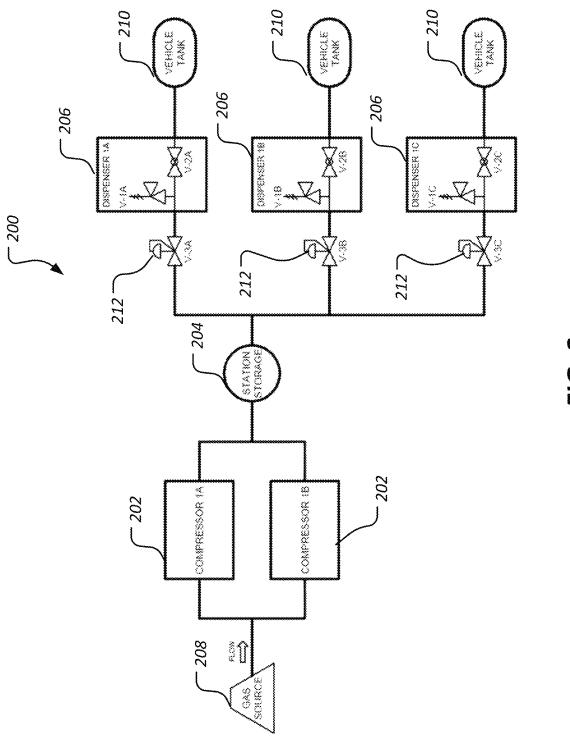
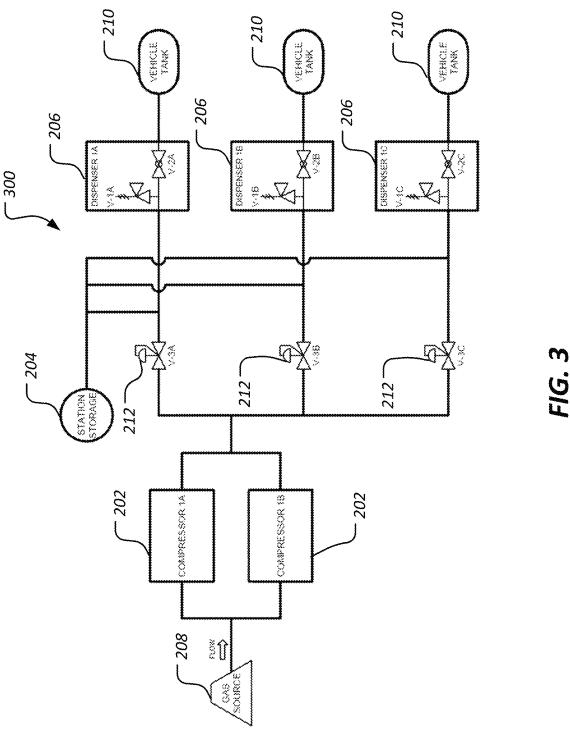
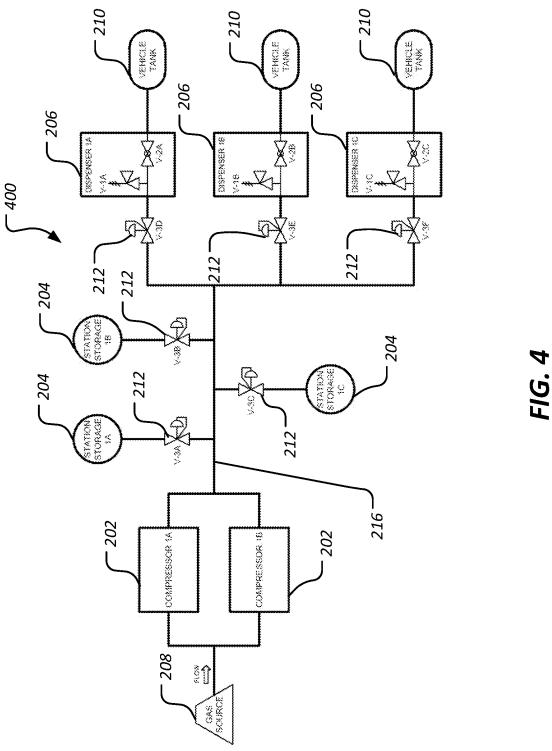
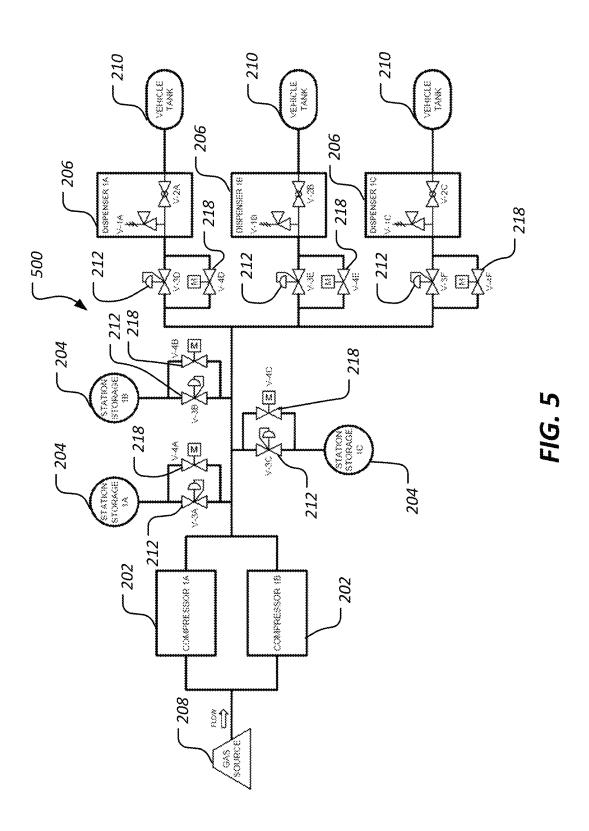
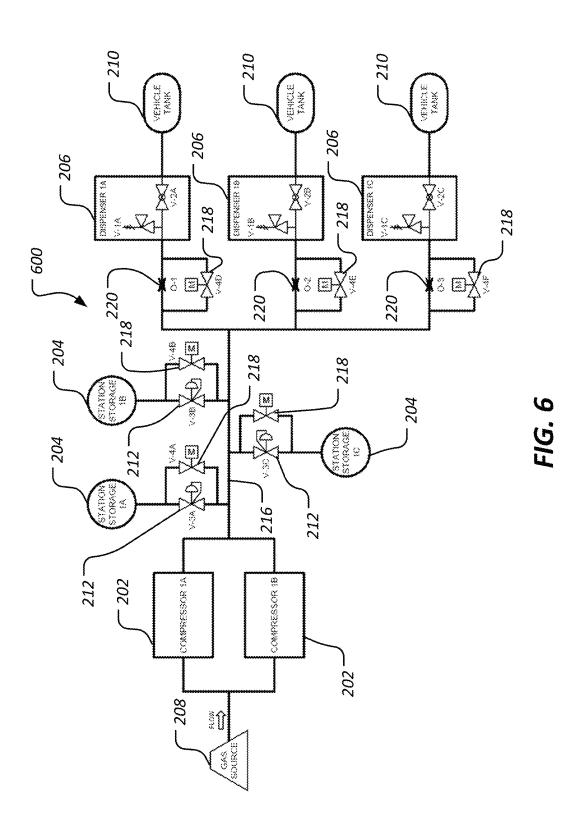


FIG. 2









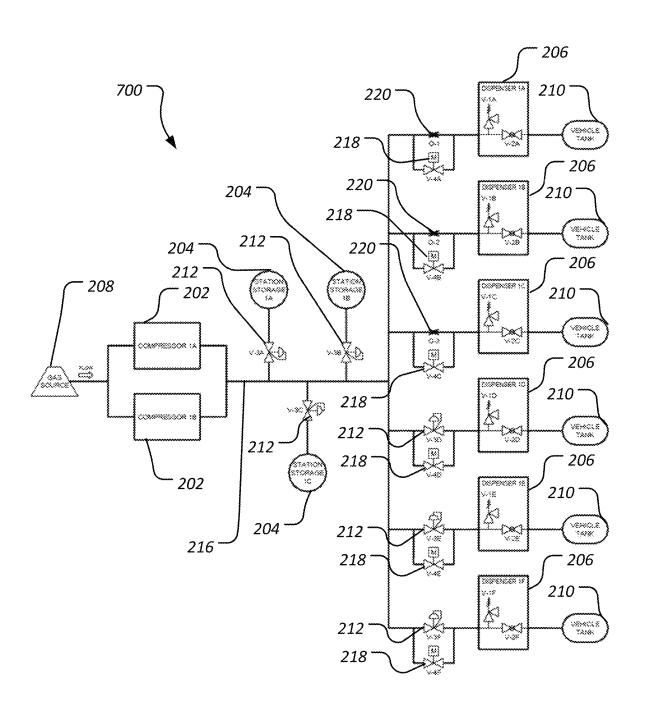
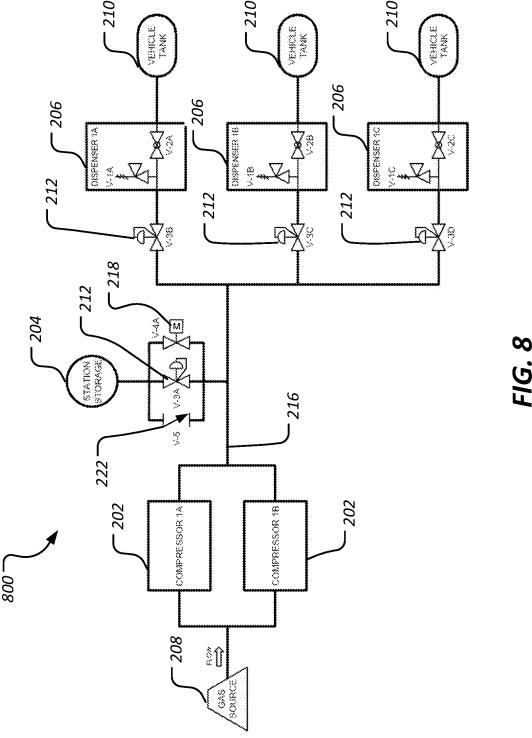
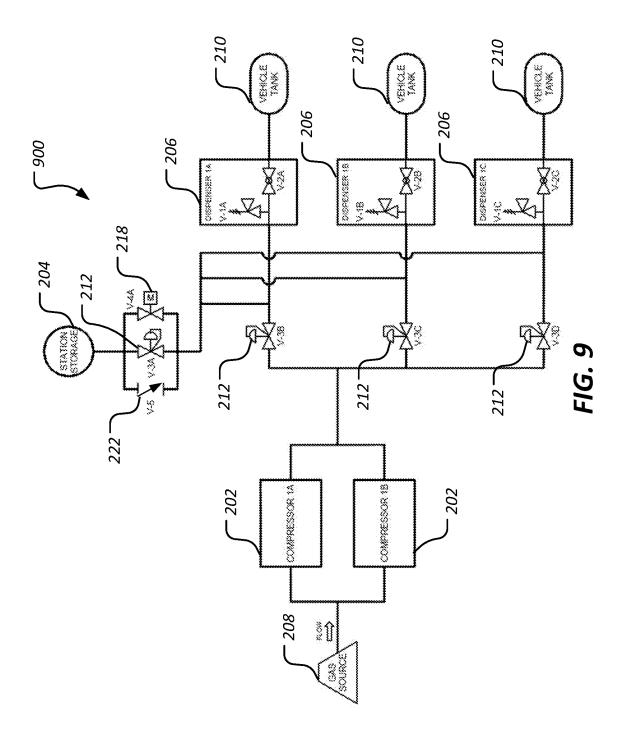
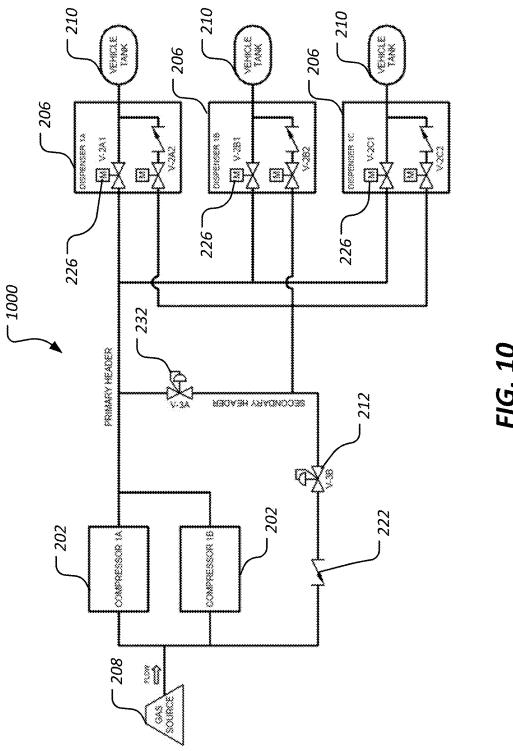
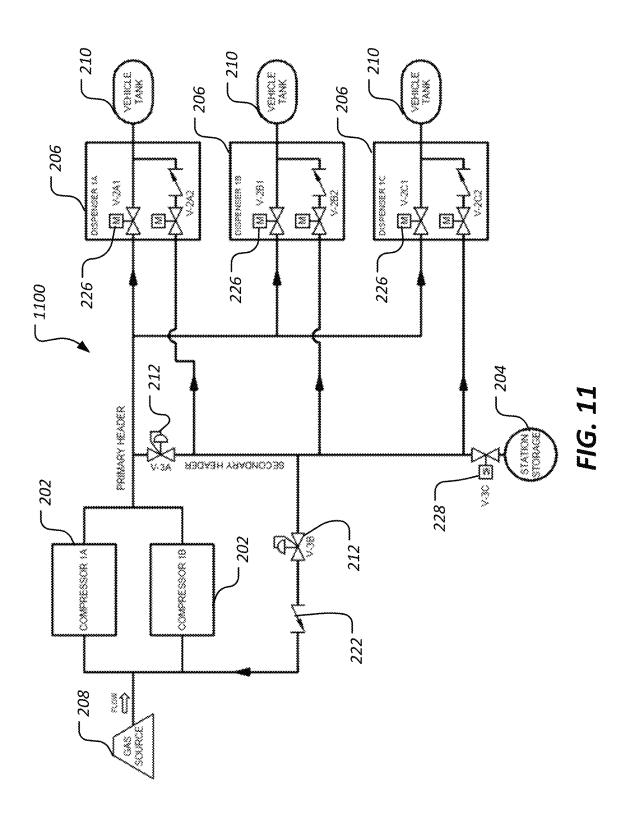


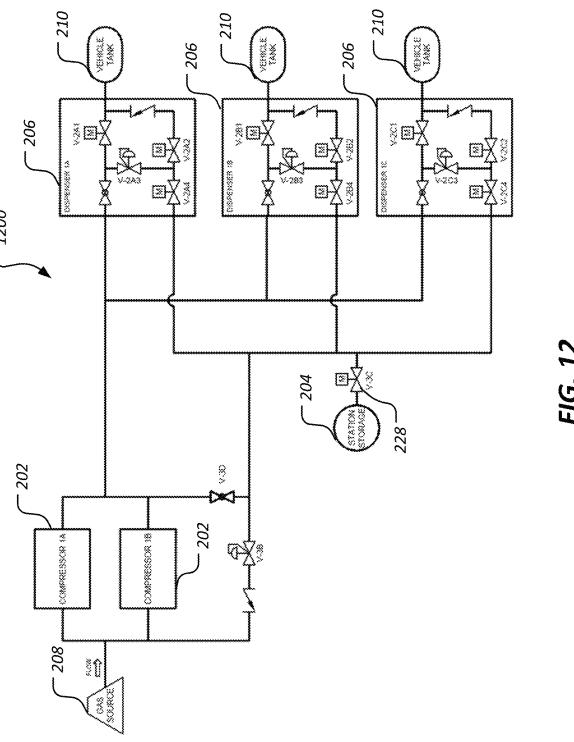
FIG. 7

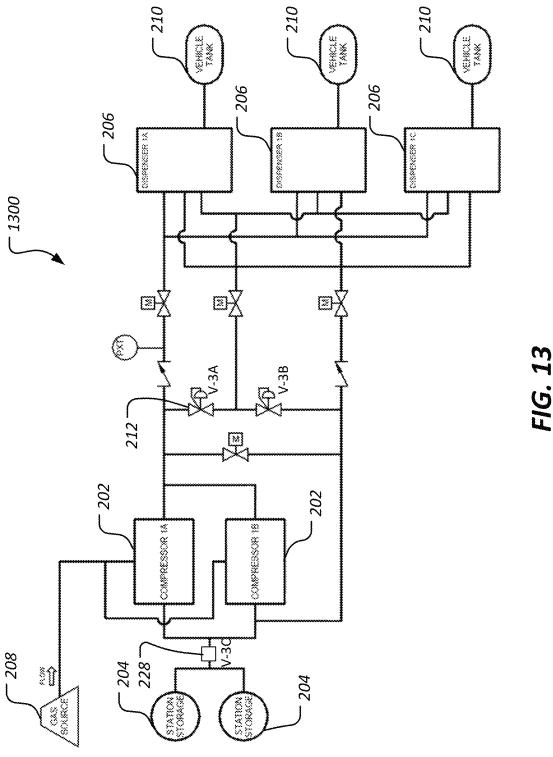


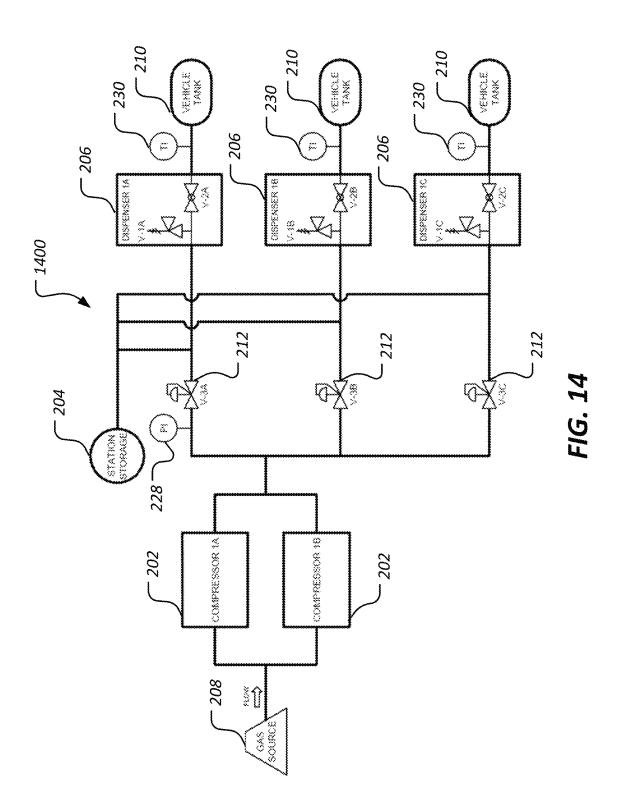












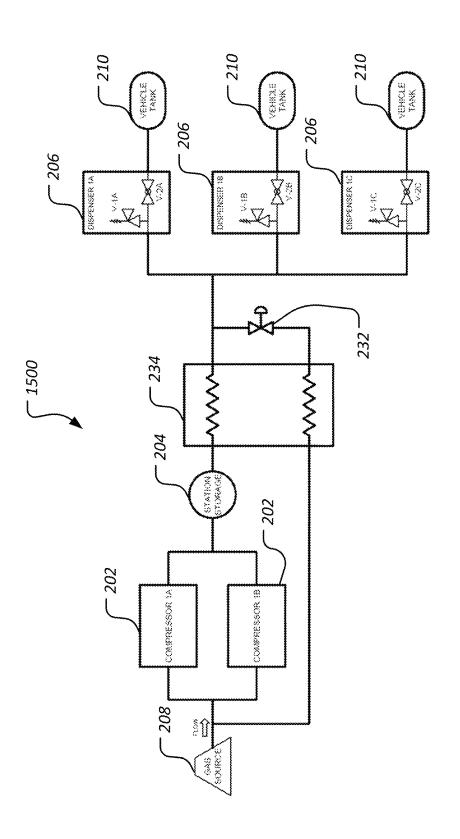
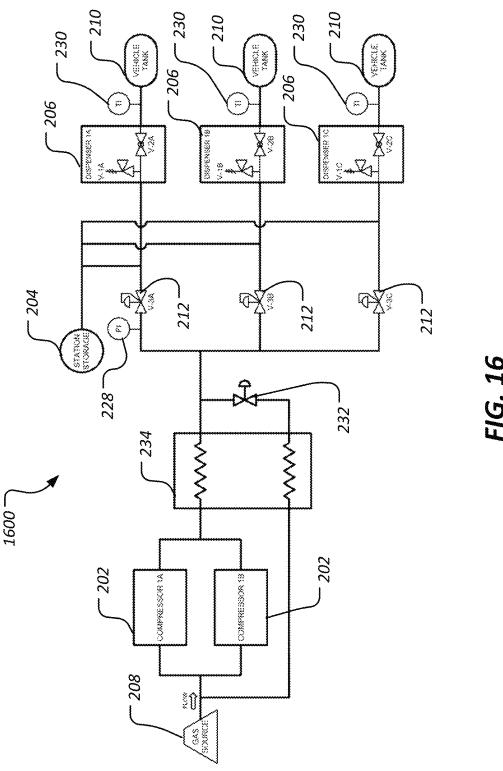


FIG. 15



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FLOW CONTROL SYSTEM

CROSS-REFERENCE TO RELATED APPLICATIONS

This patent application claims the benefit of the filing date of the U.S. Provisional Patent Application Ser. No. 62/214, 168, filed on Sep. 3, 2015 and entitled "Flow Control System," the entire content of which is hereby expressly incorporated by reference.

STATEMENT REGARDING FEDERALLY SPONSORED RESEARCH OR DEVELOPMENT

Not applicable.

REFERENCE TO A MICROFICHE APPENDIX

Not applicable.

BACKGROUND

Filling vehicle tanks with compressed natural gas (CNG) can sometimes be time consuming and there is a need for prioritized filling of selected vehicle tanks.

BRIEF DESCRIPTION OF THE DRAWINGS

- FIG. 1 is a schematic diagram of a compressed natural gas (CNG) compressed natural gas (CNG) refueling station 30 system according to an embodiment of the disclosure.
- FIG. 2 is a schematic diagram of a compressed natural gas (CNG) compressed natural gas (CNG) refueling station system according to another embodiment.
- FIG. 3 is a schematic diagram of a compressed natural gas 35 (CNG) compressed natural gas (CNG) refueling station system according to another embodiment.
- FIG. 4 is a schematic diagram of a compressed natural gas (CNG) compressed natural gas (CNG) refueling station system according to another embodiment.
- FIG. 5 is a schematic diagram of a compressed natural gas (CNG) compressed natural gas (CNG) refueling station system according to another embodiment.
- FIG. 6 is a schematic diagram of a compressed natural gas (CNG) compressed natural gas (CNG) refueling station 45 system according to another embodiment.
- FIG. 7 is a schematic diagram of a compressed natural gas (CNG) compressed natural gas (CNG) refueling station system according to another embodiment.
- FIG. 8 is a schematic diagram of a compressed natural gas 50 (CNG) compressed natural gas (CNG) refueling station system according to another embodiment.
- FIG. 9 is a schematic diagram of a compressed natural gas (CNG) compressed natural gas (CNG) refueling station system according to another embodiment.
- FIG. 10 is a schematic diagram of a compressed natural gas (CNG) compressed natural gas (CNG) refueling station system according to another embodiment.
- FIG. 11 is a schematic diagram of a compressed natural gas (CNG) compressed natural gas (CNG) refueling station 60 system according to another embodiment.
- FIG. 12 is a schematic diagram of a compressed natural gas (CNG) compressed natural gas (CNG) refueling station system according to another embodiment.
- FIG. 13 is a schematic diagram of a compressed natural 65 gas (CNG) compressed natural gas (CNG) refueling station system according to another embodiment.

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FIG. 14 is a schematic diagram of a compressed natural gas (CNG) compressed natural gas (CNG) refueling station system according to another embodiment.

FIG. **15** is a schematic diagram of a compressed natural gas (CNG) compressed natural gas (CNG) refueling station system according to another embodiment.

FIG. 16 is a schematic diagram of a compressed natural gas (CNG) compressed natural gas (CNG) refueling station system according to another embodiment.

DETAILED DESCRIPTION

Referring now to FIG. 1, a schematic diagram of embodiment of a compressed natural gas (CNG) refueling station system 100 is shown. The system 100 generally comprises a combination of at least one compressor 102, at least one station storage 104, and at least one dispenser 106. The compressors 102 are generally configured to receive natural gas from a gas source 108 and compress the natural gas to a pressure higher than the gas source 108 pressure. The CNG can be provided from the one or more compressors 102 to the station storage 104, from station storage 104 to the dispensers 106, and from the dispensers 106 to vehicle tanks 110. In alternative embodiments, the compressors 102 can be configured to selectively supply CNG directly to one or more of the dispensers 106.

The dispensers 106 require a differential pressure between the vehicle tanks 110 and the station storage 104 (or alternatively a pressure differential between the vehicle tanks 110 and the compressor 102 output pressure) to dispense CNG to the vehicle tanks 110. When multiple dispensers 106 are active (filling one or more vehicle tanks 110) the system 100 will balance the pressure drops to be the same between the CNG source relative to the dispensers 106, such as the station storage 104 or compressors 102, and the different destinations (vehicle tanks 110). This means that the CNG will flow to the lowest pressure vehicle tank 110 until that vehicle tank 110 pressure has risen up to the pressure of the next lowest vehicle tank 110 pressure. Next, CNG will flow to each of the vehicle tanks 110 until the CNG reaches the pressure of the next lowest vehicle tank 110 pressure. The end result can be that vehicle tanks 110 that begin being filled during the filling of other vehicle tanks 110 (such as in the case of later arriving vehicles), the vehicle tanks 110 of the simultaneously refilling vehicles will all finish filling at substantially the same time. For instance, consider a case where a vehicle #1 has been filling and the vehicle tank 110 of vehicle #1 is at 3000 psig and the vehicle tank 110 of vehicle #1 is considered filled when the pressure of the vehicle tank 110 of vehicle #1 reaches 3600 psig. When a vehicle #2 begins filling and has only 1500 psig in the vehicle tank 110 of vehicle #2, all of the gas that was flowing to vehicle #1 will be diverted and begin filling the vehicle tank 110 of vehicle #2. When the vehicle tank 110 for vehicle #2 reaches 3000 psig, the flow of CNG will be split between filling the vehicle tank 110 of vehicle #1 and the vehicle tank 110 of vehicle #2. The vehicle tanks 110 of both vehicles will reach 3600 psig at substantially the same time, regardless of when they began filling.

Referring now to FIG. 2, a CNG refueling station system 200 is shown. The system 200 is substantially similar to system 100 but with the addition of valves 212 (V-3A, V-3B, V-3C) disposed in fluid communication between the station storage 204 and the dispensers 206. The addition of the valves 212 can allow a more timely fill for the first vehicle described above so that the first vehicle is refilled prior to the complete refilling of the second vehicle (first in first out).

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The valves 212 can be used to control flow rate to the first vehicle, despite the arrival of subsequent vehicles and the later initiated filling of additional vehicle tanks 210. In some embodiments, the management of which vehicle tank 210 is completely filled first can comprise adding restrictions that 5 cause corresponding pressure drops in the supply lines to the dispensers 206 associated with the relatively lower priority (last in) vehicles. When only one vehicle tank 210 (such as the vehicle tank 210 associated with dispenser 206 (such as dispenser 1A)) is filling, the valve 212 (such as valve V-3A) can remain wide open to minimize pressure drop between the vehicle tank 210 and the station storage 204. When a second vehicle tank 210 (such as the vehicle tank 210 associated with dispenser 206 (such as dispenser 1B)) begins to fill, the flow rate of CNG to the vehicle tank 210 15 associated with dispenser 1A and valve V-3A would be monitored. If the flow rate to the vehicle tank 210 the vehicle tank 210 associated with dispenser 1A and valve V-3A falls below a desired flow rate, the valve 212 (such as valve V-3B) can be selectively controlled to provide the necessary above- 20 described restriction and/or pressure drop in order to maintain the desired flow rate to the vehicle tank 210 associated with dispenser 1A and valve V-3A. However, in alternative embodiments, the valves 212 can be included in one or more dispensers 206 and/or can replace an existing valve in a 25 dispenser 206.

Referring now to FIG. 3, a CNG refueling station system 300 is shown. The system 300 is substantially similar to the system 200. However, the station storage 204 is connected between the valves 212 and the dispensers 206 rather than 30 upstream of the valves 212.

Referring now to FIG. 4, a CNG refueling station system 400 is shown. The system 400 is substantially similar to the system 200. However, system 400 further comprises valves 212 (such as control valves V-3A-V-3C) disposed in fluid 35 communication between the multiple station storage tanks 204 and the compressor header 216. System 400 comprises valves 212 (valves V-3D-V-3F) disposed in fluid communication between the dispensers 206 and the compressor discharge manifold 216. As compared to system 200, the 40 addition of the control valves 212 (such as valves V-3A-V-3C) can allow a controller to push gas into the station storage tanks 204 while maintaining a constant dispensing pressure provided to the dispensers 206.

Referring now to FIG. 5, a CNG refueling station system 45 500 is shown. The system 500 is substantially similar to the system 400. However, system 500 further comprises bypass valves 218 (such as valves V-4A-V-4F) disposed as potential bypasses around the valves 212 (valves V-3A-V-3F). The bypass valves 218 can be controlled to reduce pressure drop 50 and increase flow rate in the lines in which valves 212 are disposed. In some cases, a vehicle that is in first priority (such vehicle A comprising a vehicle tank 210 associated with dispenser 206 (such as dispenser 1A), a bypass valve 218 (such as valve V-4D) can be relatively more open and/or 55 fully open to minimize pressure drop while other bypass valves 218 (such as valve V-4B and V-4C) can remain relatively more closed and/or fully closed, thereby forcing the gas to travel through the valves 212 (such as valves V-3E and V-3F). In addition, the bypass valves 218 associated 60 with the station storage tanks 204 (such as valve V-4A-V-4C) can remain open when the dispensers 206 are not actively filling vehicle tanks 210, thereby preventing a compressor 202 discharge pressure from being artificially raised while filling station storage tanks 204.

Referring now to FIG. 6, a CNG refueling station system 600 is shown. The system 600 is substantially similar to the

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system 500. However, system 600 comprises orifices 220 (such as orifices O-1-O-3) rather than the valves 212 (such as valves V-3D-V-3F). In operation while filling a vehicle tank 210 using dispenser 206 (such as dispenser 1A), considered the first priority vehicle A, the bypass valve 218 (such as bypass valve V-4D) can remain open while other bypass valves 218 (such as bypass valves V-4E and V-4F) can be closed. The above-described orifices 220 can allow a small amount of gas to pass to the dispensers 206 while forcing the majority of the gas through the open bypass valve 218 (such as bypass valve V-4D). In some cases, the orifices 220 can be utilized to avoid completely shutting off the lower priority dispensers so that automatic time outs can be avoided. The automatic time outs can cause the dispensers 206 to shut off in response to not receiving an adequate flow rate for a predetermined period of time. The orifices 220 in system 600 can be adjusted/calibrated to ensure that the dispensers 206 can remain active (avoid timing out) while they are not in first priority. In some cases, as a vehicle tank 210 fills and the associated dispenser 206 deactivates, all of the vehicles (more specifically, vehicle tanks 210) can increase one position in priority.

Referring now to FIG. 7, a CNG refueling station system 700 is shown. The system 700 is substantially similar to system 500. However, system 700 further comprises the orifices 220 disclosed above with reference to system 600. In system 700, the orifices 220 are additional to the valves 212 rather than replacements for the valves 212. System 700 comprises both control valves 212 and orifices 220 disposed between the dispensers 206 and the output lines of the compressors 202 and/or station storage tanks 204. Depending on the size/flow/importance of the dispensers 206, the valves 212 and/or orifices 220 can be selectively controlled by either controlling the valves 212 and/or the orifices 220.

Referring now to FIG. **8**, a CNG refueling station system **800** is shown. System **800** comprises multiple compressors **202** feeding into a common header **216**. The header **216** is connected via a control valve **212**, a bypass valve **218**, and check valve **222** to a bank of one or more station storage tanks **204**. The common header **216** is connected to individual valves **212** (such as V-3B-V-3D) feeding three dispensers **206**.

Referring now to FIG. 9, a CNG refueling station system 900 is shown. The system 900 is substantially similar to the system 800. However, the station storage 204 is connected between the valves 212 (such as valves V-3B-V-3D) and the dispensers 206 rather than upstream of the valves 212 (such as valves V-3B-V-3D).

Referring now to FIG. 10, a CNG refueling station system 1000 is shown. The system 1000 comprises multiple headers with a primary header configured to feed the first priority vehicle, such as a vehicle A associated with dispenser 206 (such as dispenser 1A), directly from the compressors 202 to a maximum fill pressure via dispenser actuated valves 226 (V-2A1, V-2B1 and V-2C1). Maximum fill pressure on the primary header can be maintained via control valve(s) 212 (such as valves V-3A and V-3B). Excess gas that could increase the primary header pressure above the maximum fill pressure can flow through valves 212 (such as valve V-3A) to the lower priority vehicles, such as vehicles associated with other dispensers 206 (such as dispensers 1B and 1C through valves 212 (V-2B2 and V-2C2)) on the other headers. Pressurization above the maximum fill pressure of a vehicle tank 210 is selectively prevented by a control valve 212 (such as valve V-3B) which can recycle excess gas back to the compressor 202 suction header.

Referring now to FIG. 11 a CNG refueling station system 1100 is shown. The system 1100 comprises a station storage 204 connected to lower priority headers by actuated ball valve(s) 228 (such as ball valve V-3C). Gas can free flow from station storage 204 to vehicle tanks 210 of vehicles on 5 the lower priority header(s) until pressure is equalized between station storage 204 and the lower priority header(s). Next, the station storage 204 can be isolated by selectively actuating ball valve(s) V-3C until all vehicle tanks 210 have been filled. After all vehicle tanks 210 have been filled, 10 station storage 204 can be refilled.

Referring now to FIG. 12, a CNG refueling station system 1200 is shown. System 1200 comprises control valves disposed in the dispensers 206. Selectively controlling the control valves in the dispensers can allow for coordination 15 between dispenser 206 controls, more precise control of the primary header pressure and redundancy in control valves. Further, the provision and/or selective control of the control valves in the dispensers 206 can allow for the dispensers 206 to be configured to function as a multiple header system 20 filling vehicle tanks 210 at multiple pressures from the primary header and/or to be configured to function in a manner substantially similar to the functionality of other systems disclosed above by selectively changing the order of operation of valves of the dispensers 206.

Referring now to FIG. 13, a CNG refueling station system 1300 is shown. The system 1300 comprises a station storage 204 connected to each of the dispenser headers by actuated ball valve(s) 228 (such as V-3C). Gas can free flow from station storage 204 to vehicle tanks 210 via each of the 30 header(s) until pressure is equalized between station storage 204 and the header(s) of the vehicle tanks 210 connected to the header(s). When station storage 204 pressure reaches a pressure below a full pressure, the compressor(s) 202 can start and feed gas into a high priority header. Gas can flow 35 from the high priority header into the dispenser 206 until the vehicle tank(s) 210 connected to the high priority header cannot take as much flow and/or pressure as the compressor(s) 202 supply. The excess gas will then go through valve 212 (such as valve V-3A) to the second 40 priority header. As with the high priority header, the gas will fill the vehicle tank(s) 210 connected to the second priority header until the vehicle tanks 210 cannot take as much flow and/or pressure as can go into the high priority header and second priority header. Next, gas can flow through valve 212 45 (such as valve V-3B) into the low header and/or station storage 204, thereby filling both station storages 204 and any vehicles tank(s) 210 connected to the low priority header. After all vehicle tanks 210 have been filled, station storage 204 can be refilled. In this embodiment gas can be fed to the 50 compressors 202 either from the gas source 208 or from the station storage 204. When the gas is being feed from station storage 204, gas can be recirculated from the low priority header back to the suction of the compressor(s) 202, thereby allowing the dispenser 206 to fill vehicle tanks 210 at a rate 55 slower than the output of the compressor(s) 202.

During filling of the vehicle tanks 210, the temperature in the vehicle tanks 210 will initially drop due to the gas expanding into the vehicle tank 210 (due to the Joule-Thomson effect). After the initial temperature drop, as filling 60 continues, the temperature in the vehicle tank 210 will continue to rise as the pressure differential decrease (due to less Joule-Thomson effect) and increased heat caused by the heat of compression in the vehicle tank 210 being filled. The net result is an elevated temperature (above ambient temperature) within the vehicle tank 210 once the vehicle tank 210 is full. After the filling stops, the vehicle tank 210 will

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radiate the heat to atmosphere and the temperature will return to atmospheric temperature. As a result, the pressure in the vehicle tank 210 will fall. Accordingly, vehicle tanks 210 must be "over-filled" during the filling process to ultimately result in a "full" pressure within the vehicle tanks 210 after they cool down to atmospheric temperature. In some cases, gas can be cooled gas prior to entering the vehicle tanks 210 by using external coolers and tank baths.

Referring now to FIG. 14, a CNG refueling station system 1400 is shown. System 1400 is substantially similar to system 300 and can not only provide for first-in first-out control to provide priority filling of a selected vehicle tank 210, it can also be used to cool the gas prior to entering the vehicle tank 210. System 1400 further comprises a pressure sensor 228 and temperature sensors 230. The valves 212 described above (such as valve V-3A-V-3C) can be controlled such that during the beginning of the fill, they can be modulated to maintain a selected downstream temperature by raising an upstream pressure. This will increase the pressure differential between the gas source 208 and the vehicle tank 210, thereby delivering the gas at a lower temperature.

Referring now to FIG. 15, a CNG refueling station system 1500 is shown. System 1500 is configured to cool gas by allowing expansion of the gas through an expansion valve 232, thereafter passing the cooled gas through the heat exchanger 234, and then feeding the gas back to the suction of compressors 202. The cooled gas exiting the expansion valve 232 cools the heat exchanger 234 which cools the gas prior to gas being provided to the dispensers 206.

Referring now to FIG. 16, a CNG refueling station system 1600 is shown. System 1600 is configured to cool gas by allowing expansion of the gas through an expansion valve 232, thereafter passing the cooled gas through the heat exchanger 234, and then feeding the gas back to the suction of compressors 202. The cooled gas exiting the expansion valve 232 cools the heat exchanger 234 which cools the gas prior to gas being provided to the dispensers 206. In this embodiment, the heat exchanger 234 is utilized in conjunction with the control valves 212 (such as valves V-3A-V-3C).

At least one embodiment is disclosed and variations, combinations, and/or modifications of the embodiment(s) and/or features of the embodiment(s) made by a person having ordinary skill in the art are within the scope of the disclosure. Alternative embodiments that result from combining, integrating, and/or omitting features of the embodiment(s) are also within the scope of the disclosure. Where numerical ranges or limitations are expressly stated, such express ranges or limitations should be understood to include iterative ranges or limitations of like magnitude falling within the expressly stated ranges or limitations (e.g., from about 1 to about 10 includes, 2, 3, 4, etc.; greater than 0.10 includes 0.11, 0.12, 0.13, etc.). For example, whenever a numerical range with a lower limit, R₁, and an upper limit, R_u, is disclosed, any number falling within the range is specifically disclosed. In particular, the following numbers within the range are specifically disclosed: $R=R_1+k*(R_{1/2}-R_{1/2}+R_{1/2$ R_1), wherein k is a variable ranging from 1 percent to 100 percent with a 1 percent increment, i.e., k is 1 percent, 2 percent, 3 percent, 4 percent, 5 percent, . . . 50 percent, 51 percent, 52 percent, ..., 95 percent, 96 percent, 97 percent, 98 percent, 99 percent, or 100 percent. Moreover, any numerical range defined by two R numbers as defined in the above is also specifically disclosed. Use of the term "optionally" with respect to any element of a claim means that the element is required, or alternatively, the element is not 7

required, both alternatives being within the scope of the claim. Use of broader terms such as comprises, includes, and having should be understood to provide support for narrower terms such as consisting of, consisting essentially of, and comprised substantially of. Accordingly, the scope of protection is not limited by the description set out above but is defined by the claims that follow, that scope including all equivalents of the subject matter of the claims. Each and every claim is incorporated as further disclosure into the specification and the claims are embodiment(s) of the present invention.

What is claimed is:

- 1. A compressed natural gas (CNG) refueling station system, comprising:
 - a compressor;
 - a first dispenser configured to selectively receive CNG from the compressor;
 - a second dispenser configured to selectively receive CNG from the compressor;
 - a first valve disposed in fluid communication between the 20 compressor and the second dispenser; and
 - a heat exchanger comprising a first CNG path and a second CNG path, the first CNG path being disposed between the first valve and the compressor and the second CNG path being disposed between an output of 25 the first CNG path and the compressor;
 - wherein the first valve is selectively operable to maintain at least one of a predetermined CNG flow rate and a predetermined CNG pressure supplied to the first dispenser.
- 2. The CNG refueling station system of claim 1, further comprising:
 - a second valve disposed between the first valve and the second dispenser.

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- 3. The CNG refueling station system of claim 2, further comprising:
 - a first bypass valve connected upstream relative to the first valve and connected downstream relative to the first valve.
- **4**. The CNG refueling station system of claim **3**, further comprising:
 - a second bypass valve connected upstream relative to the second valve and connected downstream relative to the second valve.
- 5. The CNG refueling station system of claim 2, further comprising:
 - a second bypass valve connected upstream relative to the second valve and connected downstream relative to the second valve.
- **6**. The CNG refueling station system of claim **2**, further comprising:
 - a check valve connected upstream relative to the first valve and connected downstream relative to the first valve.
- 7. The CNG refueling station system of claim 1, further comprising:
 - an expansion valve disposed between the output of the first CNG path and an input of the second CNG path.
- **8**. The CNG refueling station system of claim **1**, further comprising:
 - a pressure sensor configured to measure a pressure of CNG between the first valve and an input to the compressor; and
 - a first temperature sensor configured to measure a temperature of CNG being provided by the second dispenser.

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